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Abstract	Floating offshore wind energy is a new form of marine renewable energy which is attracting a great deal of attention worldwide. However, the concepts of floating offshore wind turbines (FOWTs) are still in early stages of development and their failure properties are not yet fully understood. Compared to bottom-fixed wind turbines, FOWTs are subject to more extreme environmental conditions and significant mechanical stresses which may cause a higher degradation rate and shorter mean-time-to-failure for components/structures. To fill the research gap, this paper aims to conduct qualitative and quantitative failure studies on an OC3 spar-type FOWT platform with 3 catenary mooring lines. The failure analyses are performed based on two well-established reliability engineering methodologies, namely, fault tree analysis (FTA) and failure mode and effects analysis (FMEA). The most critical FOWT components are prioritized according to their failure likelihood as well as the risk-priority-number. Our results show a good agreement between the two methods with regard to failure criticality rankings. However, some differences between the results are also observed that are attributed to the difference between FTA and FMEA methodologies as the former incorporates the causes of various failure modes into analysis, whereas the latter is mainly adopted for a single random failure analysis. The results obtained from the FMEA study for the FOWT system will also be compared with those reported for bottom-fixed offshore wind turbines and some interesting conclusions are derived.		
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² Failure analysis of spar buoy floating offshore wind turbine systems

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6 Abstract

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7 Floating offshore wind energy is a new form of marine renewable energy which is attracting a great deal of attention world-8 wide. However, the concepts of floating offshore wind turbines (FOWTs) are still in early stages of development and their 9 failure properties are not yet fully understood. Compared to bottom-fixed wind turbines, FOWTs are subject to more extreme 10 environmental conditions and significant mechanical stresses which may cause a higher degradation rate and shorter mean-11 time-to-failure for components/structures. To fill the research gap, this paper aims to conduct qualitative and quantitative 12 failure studies on an OC3 spar-type FOWT platform with 3 catenary mooring lines. The failure analyses are performed 13 based on two well-established reliability engineering methodologies, namely, fault tree analysis (FTA) and failure mode and 14 effects analysis (FMEA). The most critical FOWT components are prioritized according to their failure likelihood as well 15 as the risk-priority-number. Our results show a good agreement between the two methods with regard to failure criticality 16 rankings. However, some differences between the results are also observed that are attributed to the difference between FTA 17 and FMEA methodologies as the former incorporates the causes of various failure modes into analysis, whereas the latter is 18 mainly adopted for a single random failure analysis. The results obtained from the FMEA study for the FOWT system will 19 also be compared with those reported for bottom-fixed offshore wind turbines and some interesting conclusions are derived.

Keywords Failure analysis · Floating offshore wind turbine (FOWT) · Materials and structures · Mooring system · Fault
 tree analysis (FTA) · Failure mode and effects analysis (FMEA)

²² Introduction

23 The development of renewable wind energy was initially 24 stimulated in the 1970s due to the increase in fossil fuel 25 prices as well as rising concerns about energy security. It 26 was supported later on by the need to reduce greenhouse 27 gas emissions and the potential to mitigate the effects of 28 climate change [1]. Currently, there are various wind tur-29 bine models with rated power ranging from 100 KW up to 30 15 MW that are manufactured to convert wind energy into 31 electrical energy in an eco-friendly way. The wind turbines 32 are installed either onshore (on land) or offshore (at sea). 33 Offshore wind turbines have gained more attention than 34 onshore wind turbines across the world in recent years. This 35 is mainly because the offshore wind resources are abundant, 36 stronger, and blow more consistently than land-based wind resources. In addition, offshore wind turbines are more visually appealing and less noisy than onshore wind turbines [2].

Currently, most offshore wind farms have been constructed using conventional fixed-bottom substructure technologies (such as monopile, tripod and jacket) within a few miles of the coastline in shallow waters (up to 50 ms water depth) [3]. In order to take advantage of the greater wind resources and wider open spaces further away from the coast, offshore wind turbines require to be sited in regions of deeper water. Floating offshore wind technology is regarded as an ideal solution for locations at water depths between 50 and 200 m [4]. Floating offshore wind energy is anticipated to have a significant growth in the near future. Out of all the continents in the world, Europe is at the forefront of floating offshore wind technology in the world. Figure 1 shows the ongoing and forecasted capacity of offshore floating wind installations in different parts of the world, including Europe, Asia, and Americas. As shown in the figure, the global installed capacity of floating offshore wind energy is anticipated to reach about 13 GW by 2030.

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Fig. 1 The predicted offshore floating wind energy capacity in MW [5]

Up-to-date, a number of floating offshore wind tech-57 nologies such as Hywind, WindFloat and Floatgen have 58 59 been prototyped and the industry has made good progress with pilot programs to test these technologies in controlled 60 environments [6]. Many research programs have aimed at 61 62 optimizing the manufacturing and maintenance processes to improve floating offshore wind energy generation; for 63 example, the readers can refer to [7-11]. Hywind is the first 64 megawatt-scale floating offshore wind project which was 65 commissioned by Statoil (currently Equinor) in October 66 2017. The wind farm is located 25 km off the coast of Aber-67 deenshire in Scotland. It consists of five 6 MW floating wind 68 turbines which provide power to more than 20,000 house-69 holds. Each of the wind turbines is mounted on a spar-buoy 70 type platform which is moored by three catenary chains to 71 the seabed [12]. 72

In spite of all recent developments, the technologies of 73 floating offshore wind turbines (FOWTs) are not yet mature 74 enough and their failure properties are not yet fully under-75 stood [13]. The future growth of floating wind power is 76 heavily reliant on the failure performance of systems and 77 their components throughout the lifecycle. Compared to 78 bottom-fixed wind turbines, FOWTs are subject to more 79 severe loads caused by wind, waves, current, tides, etc. The 80 severe loading conditions in deep waters can lead to struc-81 tural defects and an associated higher failure risk and/or 82 83 shorter mean-time-to-failure for components. An unexpected failure in FOWTs may result in undesirable consequences 84 such as reduction in electricity production, loss of asset, or 85 86 even more catastrophic events such as personal injuries or loss of life of personnel. Early detection of potential failures 87 and taking appropriate remedial measures for eliminating 88 their causes can help wind farm managers save operation 89 and maintenance (O&M) costs [14]. 90

A brief review of the literature shows that very few
 studies have been carried out to evaluate various failure
 mechanisms associated with FOWTs and their supporting

structures. Guo et al. [15] conducted a qualitative fault tree 94 analysis (FTA) for FOWTs and showed that mooring system, 95 lubrication system of gearbox, cooling system, and yaw sys-96 tem were among the riskiest components. A dynamic FTA 97 study for FOWTs was also conducted by Zhang et al. [16]. 98 The authors took all the relationships between modules and 99 failure mechanisms into consideration and based on sys-100 tem grading they derived a series of high-risk factors that 101 resulted in failure of the whole system. Kang et al. [17] per-102 formed a failure mode and effects analysis (FMEA) study on 103 FOWTs and then compared the results of their analysis with 104 those obtained by a reliability index vector (RIV) method. 105 Kang et al. [18] adopted the FTA method for qualitative and 106 quantitative failure analyses of semi-submersible FOWTs. It 107 was shown that marine conditions, especially the salt-spray 108 and high wind speed have the highest impact on FOWT per-109 formance. More recently, Li et al. [19] extended the conven-110 tional FMEA methodology to analyze the failures of support 111 structures in FOWTs. Based on the analysis, some sugges-112 tions were made on maintenance actions aiming at ensuring 113 the safe and economic operation of support structures. 114

From the reviewed studies, it is evident that there is so 115 far no study in the literature comparing the performance of 116 various methodologies for failure analysis of FOWT tech-117 nologies. A comparative study will be useful to decide on 118 the most efficient way of analyzing damage mechanisms or 119 failure modes of the FOWT components. In addition to this, 120 the existing studies do not evaluate the severity of failure 121 modes associated with underwater components of FOWTs, 122 including the platform, mooring system, and connection 123 cables. In order to overcome these gaps, this study aims to 124 provide a comparative analysis between 2 well-established 125 reliability engineering methodologies, namely FTA and 126 FMEA for an OC3-Hybrid spar-type FOWT system. Such 127 comparative analysis will help operators and asset manag-128 ers better understand the performance of different failure 129 assessment methodologies and choose the method that is 130

more appropriate for them. Our analysis covers all major 131 mechanical, electrical, and structural subassemblies of the 132 system, including floating platform, mooring lines, tower 133 structure, pitch and hydraulic system, blade control system, 134 gearbox, generator, etc. Failure information of the FOWT 135 subassemblies is collected from previous studies, indus-136 try databases such as 4C Offshore, as well as the reports 137 published by floating wind power companies such as Equi-138 nor, BW Ideol, Principle Power. The most critical FOWT 139 subassemblies are identified and ranked according to their 140 failure likelihood and also risk priority number (RPN). The 141 results obtained from both FTA and FMEA methods are 142 compared and analyzed. Our findings reveal a good agree-143 ment between the 2 methods with regard to failure criticality 144 rankings. However, some differences between the results are 145 also observed that are attributed to the difference between 146 FTA and FMEA methodologies as the former incorporates 147 the causes of various failure modes into analysis whereas the 148 latter is mainly adopted for a single random failure analysis. 149 The results obtained from the FMEA study are also com-150 pared with those reported for bottom-fixed offshore wind 151 farms. The RPN rankings from present work show good 152 agreement with the literature. 153

The remainder of this article is organized as follows Sec-154 tion 2 presents a brief overview of FOWT technologies and 155 failure analysis methodologies so as to set the background 156 for the main contribution of the paper. Section 3 describes 157 the FTA and FMEA methodologies adopted for failure 158 analysis of the OC3-Hywind spar-type FOWT technology. 159 Section 4 presents the results and discusses the findings. 160 Section 5 concludes the study with suggestions on future 161 areas of research. 162

163 Research background

164 FOWT technology

The potential for floating offshore wind power is signifi-165 cantly greater than conventional bottom-fixed offshore wind 166 power. A floating wind turbine is a wind turbine mounted on 167 a floating platform that is connected to the seabed by moor-168 ing lines. Therefore, the platform and mooring system are 169 crucial parts of a FOWT technology. The FOWT platforms 170 are typically categorized into 3 major concepts, including: 171 spar-buoy, semi-submersible, and tension-leg. These 3 con-172 cepts are shown in Fig. 2 and are explained briefly in the 173 following sections. 174

This study focuses on a floating wind turbine concept based on an OC3-Hywind spar type of platform that is moored to the seabed with three anchor piles. The sparbuoy platform is characterized by small plane area and large cylindrical mass below the water surface, a design that is



Fig. 2 Floating offshore wind platforms: spar-buoy (right), semi-submersible (center), and tension-leg (right) (https://windeurope.org/)

favorable for deep water applications. This concept allows 180 installations in water depths of greater than 100 m [4]. The 181 top section of the structure is lighter than the bottom sec-182 tion, which raises the center of buoyancy. In order to achieve 183 static stability, it uses ballast weights that are placed low in 184 the buoy, making the center of gravity lower than the center 185 of buoyancy. Therefore, it provides high resistance to the 186 rotational motions of pitch and roll. Spar-buoy platforms 187 are usually made from either concrete or steel, while the 188 ballast weights can be water or solid material. Mooring lines 189 with embedded anchors to the seabed help not only to keep 190 the structure in place but also contribute towards minimiz-191 ing surge and sway motions. Typical mooring line materials 192 include fiber ropes, steel cables or anchor chains. 193

Over the past decade, extensive research has been per-194 formed to evaluate the mechanical performance of spar-195 buoys as FOWT platforms. Jonkman et al. [20] and Jonkman 196 [21] reported the mechanical properties of an OC3-Hywind 197 FOWT system carrying the NREL 5 MW reference wind 198 turbine. Karimirad and Moan [22] investigated the feasibility 199 of deploying spar-type floating wind platforms at moderate 200 water depth. The authors used the aeroelastic code HAWC2 201 (Horizontal Axis Wind turbine simulation Code 2nd gen-202 eration) for calculating the wind turbine's response in time 203 domain. This code was originally developed by the aeroelas-204 tic design research programme at Risø DTU in Denmark. In 205 another study, Karimirad and Moan [23] compared the power 206 performance, structural integrity, and dynamic responses of 207 2 spar-based FOWT platforms using different codes such as 208 SIMO-RIFLEX and TDHMILL3D. The platforms included 209 one called shortspar and another called deepspar, which were 210 deployed, respectively, in moderate and large water depths. 211 Nematbakhsh et al. [24] proposed a nonlinear computational 212 model, based on the Navier-Stokes equations, to simulate the 213 motion of a 5 MW spar buoy floating wind turbine in extreme 214

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sea states including waves over 17 m height. Chen et al. [25] 215 conducted a series of comparisons on dynamics characteristics 216 of spar-buoy and semi-submersible floating wind turbines. It 217 was found that the spar-buoy floating wind turbine is more sen-218 sitive to wind loading, whereas the semi-submersible floating 219 wind turbine is more sensitive to wave loading. Sultania and 220 Manuel [26] proposed two-dimensional and three-dimensional 221 inverse first-order reliability methods for a spar-supported 222 floating offshore 5 MW wind turbine under variable environ-223 mental and load conditions. Ahn and Shin [27] developed an 224 OC3 spar-buoy floating wind turbine model moored by a 3-leg 225 catenary spread mooring system with a delta connection. They 226 verified the results obtained from numerical simulation tools 227 with the performance of OC3-Hywind platforms in combined 228 wave and wind environments. Lin et al. [28] proposed a simu-229 lation model to estimate dynamic responses of spar buoy and 230 tension-leg floating offshore wind turbines. The study devel-231 oped a modular system based on MATLAB SIMULINK in 232 combination with a boundary element method (BEM) solver 233 and visualization software ParaView. Bashetty and Ozcelik 234 [29] reviewed the historical developments and progresses in 235 the design of different types of FOWT platforms including 236 spar type, semisubmersible, and tension leg platforms. The 237 dynamics characteristics of the FOWT platforms for a single 238 turbine and multiple turbines under various operating environ-239 mental conditions were also discussed. 240

241 Failure analysis methodologies

242 Fault tree analysis (FTA)

FTA is one of the most popular and effective methods for 243 failure analysis of onshore/offshore wind turbines [30]. It is 244 a top-down, deductive failure analysis method through which 245 undesired states of a system can be identified. The method uses 246 a logic diagram which begins with an undesired top event and 247 then works backward toward identifying different sub-events 248 that contribute to the top event [31]. The sub-events are con-249 nected via logic symbols (known as gates) which show the 250 relationship between successive levels of the tree. The most 251 common symbols and logic gates used in FTA are shown in 252 Fig. 3. AND gate means that the output event will occur only 253 if all the input events occur simultaneously, whereas OR gate 254 means that the output event will occur if at least one of the 255 input events occurs. 256

FTA can also be used to determine the likelihood of 257 occurrence of the top event. However, extensive calcula-258 tions are required and sometimes discrepancies may exist 259 between actual failure in practice and reliability estima-260 tions. The probability of a gate's output event depends on 261 the type of the gate as well as input event probabilities. 262 An AND gate represents the intersection of the events 263 attached to the gate. Assuming A and B are 2 independent 264 events, then the probability of their intersection is just the 265 product of their probabilities. Thus, 266

$$P(A \text{ AND } B) = P(A \cap B) = P(A) \times P(B)$$
 (1)
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On the other hand, an OR gate corresponds to set union and thus the probability of the OR gate output is given by:

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$$P(A \text{ OR } B) = P(A \cup B) = P(A) + P(B) - P(A \cap B)$$
 (2)
(2) 272

Since failure probabilities on fault trees often tend to273be small (< 0.01), P (A AND B) usually becomes a very274small error term, and the output of an OR gate may be275conservatively approximated by using an assumption that276the inputs are mutually exclusive events:277

$$P(A \cap B) \approx 0P(A \text{ OR } B) \approx P(A) + P(B)$$
 (3) ²⁷⁸
₂₇₉

Failure mode and effects analysis (FMEA)

Failure mode and effects analysis is one of the most popu-281 lar failure analysis methods in the wind energy industry 282 (e.g., [32, 33]). This method involves creating a series of 283 linkages between failure modes of a system, their effects 284 on the system performance, and the underlying causes of 285 the failure. In this method, the criticality of a failure is 286 assessed based on an index called the risk priority num-287 ber. The RPN is obtained by multiplying the scores of 3 288 factors, namely, the probability of failure occurrence (O), 289 severity of failure consequence (S), and probability of not 290 detecting the failure (D). In the wind energy industry, O, S 291 and D are evaluated using four-point scales given in sum-292 mary in Tables 1, 2 and 3 as proposed in [34]. 293

According to the above rating scales for O, S and D, the RPN value for each failure mode will range between 295 1 and 200 (= 5 × 4 × 10). The FMEA method is most beneficial when carried out as an iterative process during the preliminary design stages, allowing for improvements and reliability monitoring. 294



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Table 1Four-point scales foroccurrence of failure

Description	Criteria
Level E (extremely unlikely)	The probability of occurrence is < 0.001
Level D (remote)	The probability of occurrence is > 0.001 but < 0.01
Level C (occasional)	The probability of occurrence is > 0.01 but < 0.10
Level A (frequent)	The probability of occurrence is > 0.10
	Description Level E (extremely unlikely) Level D (remote) Level C (occasional) Level A (frequent)

Table 2Four-point scales forseverity of failure

Rank	Description	Criteria
1	Category IV (minor)	Electricity can be generated but an urgent repair is required
2	Category III (marginal)	Reduction in ability to generate electricity
3	Category II (critical)	Loss of ability to generate electricity
4	Category I (catastrophic)	Major damage to the wind turbine

Table 3Four-point scales fordetection of failure

Rank	Description	Criteria
1	Almost certain	Current monitoring methods almost always will detect the failure
4	High	Current monitoring methods will highly likely detect the failure
7	Low	Current monitoring methods will low likely detect the failure
10	Almost impossible	No known monitoring method is available to detect the failure

300 Failure analysis of FOWT

Previous studies about the failure analysis of FOWTs 301 are all focused on semi-submersible floating platforms. 302 In this study, a failure analysis on an OC3-Hywind spar-303 type FOWT model is performed using the FTA and FMEA 304 305 methodologies. The FOWT model was designed to support a 5 MW NREL offshore baseline wind turbine mounted on 306 an OC3-Hywind spar platform [20]. The FOWT is moored 307 by a system of three catenary lines to the seabed. The 308 lines are attached to the platform via a delta connection to 309 increase the yaw stiffness of mooring lines. 310

Since the available failure data for the OC3-Hywind 311 spar-type FOWT model was limited, the failure infor-312 mation for the analysis was obtained from the published 313 industry reports (mainly by Carbon Trust, Equinor, Ørsted, 314 and BW Ideol) as well as expert opinions. Our analysis 315 focused on estimating the probability of failure of the 316 whole system as well as each of the sub-systems/com-317 ponents. The subsystems/components considered in this 318 study include: spar-buoy platform, mooring system, tower 319 structure, electronic components, rotor blades, yaw sys-320 321 tem, drivetrain system (consisting of gearbox, generator and the brake unit), and pitch and hydraulic system. The 322 software tool used for this study is PTC Windchill (for-323 merly Relex), version 11.0 (https://support.ptc.com/produ 324 cts/windchill/quality/). This software can be used for a 325

variety of purposes such as reliability prediction, FTA, 326 Markov modeling and Weibull analysis as well as drawing 327 reliability block diagrams (RBDs). 328

FTA of OC3 spar-type FOWT

The fault tree diagram of the OC3-Hywind spar-type FOWT330model is shown in Fig. 4. As the subassemblies/components331are connected to each other in series, an OR gate was used332to connect the fault categories to the top event. In what fol-333lows, the fault tree diagrams of individual sub-assemblies334are constructed.335

Spar-buoy platform

The spar-buoy platform is well-known for its inherent stabil-337 ity due to its low center of gravity. The fault tree diagram for 338 a spar-buoy floating platform is depicted in Fig. 5. As can 339 be seen, the spar-buoy floating platform may fail due to 5 340 known basic events: mooring system failure, strong wind/ 341 wave, typhoon, crash with vessels and biological collision. 342 If the mooring system fails, the floating platform will still 343 stay afloat albeit with the risk of wandering further from its 344 site. However, if harsh environmental conditions like strong 345 winds or high waves occur at the same time they could cause 346 the structure to capsize. Thus, the mooring system failure 347 and strong wind/wave were connected with each other via 348 an AND gate. The floating platform may also be damaged 349

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Fig. 5 Fault tree diagram of a spar-buoy floating platform

 Table 4
 Failure rates of the basic events for a spar-buoy platform

Intermediate / basi	Failure rate (h ⁻¹)	
Capsize	Mooring system failure	2.04×10^{-4}
	Strong wind/wave	5.00×10^{-5}
External objects	Typhoon	1.00×10^{-4}
	Crash with vessels	1.00×10^{-6}
	Biological collision	5.00×10^{-6}

by external factors including typhoons, crash with vessels or
biological collision. These factors were therefore connected
via an OR gate. The rates of the failure causes for an OC3
Hywind spar-buoy floating platform have been reported in
[16] and [18] and are given in Table 4.

355 Mooring system

The mooring system keeps the position of the floating platform within an allowable region and avoids the drift caused by wind, current and hydrodynamic forces. The fault tree diagram for mooring system is constructed by dividing the

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system into its constituent parts, e.g., mooring lines, fairlead,
anchor, etc. The failure of either of these parts would cause360the mooring system to fail. Thus, the basic events are linked
with each other using an OR gate, as shown in Fig. 6.363

As can be seen, the spar-buoy mooring system may 364 fail due to nine known basic causes, namely, mooring line 365 fatigue, chain corrosion, abnormal stress, friction chain 366 wear, transitional chain wear, poor operation environment, 367 insufficient emergency measures, fairlead fatigue, fairlead 368 corrosion and anchoring failure. Even though the anchoring 369 failure is considered as one of the major failure modes for a 370 spar-buoy mooring system, due to insufficient data it is not 371 expanded further in this study. Table 5 gives the rates of the 372 failure causes for a spar-buoy mooring system. 373

Tower structure

The tower structure is considered as one of the most important components of FOWTs, because any damage to the tower will put the entire system in jeopardy. The fault tree diagram for a wind turbine tower structure is represented in Fig. 7. 379

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Anchor failure

As can be seen, all the intermediate and basic events 380 are connected to the top event via an OR gate, mean-381 ing that if either of these events occurs it will lead to 382 failure of the entire tower system. Welding defects may 383 occur either during manufacturing process or later dur-384 ing operation phase. External damages are considered 385 as another reason for the failure of the tower structure. 386 These damages include: lighting strike, heavy storm and 387 strong wind/wave. Table 6 gives the rates of the failure 388 root causes for a wind turbine tower structure. 389

Electrical components

The fault tree diagram for electronic components of a wind 391 turbine system is shown in Fig. 8. As can be seen, the basic 392 failure events were categorized into 2 types: mechanical 393 faults and electrical faults. The corrosion due to moisture 394 and salty atmosphere, presence of dirt, and damage in ter-395 minals were identified as the main reasons for mechanical 396 faults, whereas the electrical faults were caused by short 397 circuit, open circuit, and gate drive circuit. 398

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 1.80×10^{-5}



Table 6 Failure rates of the basic events for a wind turbine tower structure

1.10×10^{-5}
5.00×10^{-6}
7.00×10^{-6} 5.50×10^{-5} 5.00×10^{-5}
7.00×10^{-6}

Rotor blades

In order to draw the fault tree diagram for rotor blades, two separate subtrees for blade structural failure and the rotor system failure were constructed and connected together via an OR gate. The fault tree diagram for the rotor blades system is shown in Fig. 9. The subtree diagrams for the blade structural failure and rotor system failure are shown in Figs. 10 and 11, respectively.

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Fig. 8 Fault tree diagram of electrical components

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Fig. 9 Fault tree diagram of a wind turbine rotor blades system

As can be seen in Fig. 10, the structural failures in wind 407 turbine blades may occur either due to edge damage or 408

shell damage. FOWTs are often exposed to harsh environ-409 mental conditions and therefore wind turbine blades are 410 susceptible to natural phenomena such as lightning strikes. 411 Erosion, cracking and delamination of the composite mate-412 rial are also primary events that can result in blade failure. 413

As Fig. 11 shows, the three principal events that can 414 trigger the rotor system failure are abnormal vibration, 415 rotor bearings damage and rotor hub fault. Rotor bear-416 ings can fail as a result of abrasive wear, corrosion, pit-417 ting or insufficient lubrication. Failure of the rotor hub on 418 the other hand can occur as a result of cracks on the hub, 419 surface roughness, mass imbalance of the blades and pitch 420 maladjustment. Major factors that contribute to the occur-421 rence of these events are closely related to environmental 422 conditions and salty air [16]. 423



Fig. 11 Fault tree diagram of a wind turbine rotor system

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The yaw system adjusts the orientation of the wind turbine 425 426 rotor towards the wind. Load variations due to wind speed can affect the yaw system and put the wind turbine at risk. 427 The yaw system is susceptible to damages mainly because 428 of the fluctuation and change in rotor torque during yaw-429 ing. The fluctuation in loads excites the whole system with 430 vibration and will therefore cause some damage to the wind 431 turbine. The fault tree diagram of a wind turbine yaw system 432 is represented in Fig. 12. 433

434 Drivetrain system

To draw the fault tree diagram for drivetrain system, three separate subtrees for gearbox, generator and brake unit failures were constructed. The fault tree diagram of the drivetrain system is shown in Fig. 13. The gearbox, generator and

439 brake unit are known as the most important components in

drivetrain and the failure of any of these components would

441 lead directly to the failure of drivetrain system as seen in

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the fact that these 3 components were connected to the top event via an OR gate.

The fault tree diagram of a wind turbine generator system444is represented in Fig. 14. The rates of the failure causes for a445wind turbine generator system were collected from different446references, e.g., [18, 35, 36]. This information is reported447in Table 7.448

Mechanical and electrical failures are the main contribu-449 tors to the generator failure. Mechanical failures may occur 450 due to either potential damage to generator bearings or fail-451 ure of rotor or stator components. Asymmetry, structural 452 deficiency or any kind of abnormal vibration due to external 453 factors are the basic events causing severe damage to genera-454 tor bearings, while overheating and broken bars are known 455 as the major causes of rotor and stator failures. For electrical 456 failures, the two basic events considered are wire fault and 457 synchronization failure. It should be noted that synchroniza-458 tion failure can normally be considered as a root cause for 459 the rotor and stator components failure, but in this study, it 460 has been considered as an electrical cause and hence it was 461 analyzed separately. 462



Fig. 12 Fault tree diagram of a wind turbine yaw system

Fig. 13 Fault tree diagram of a wind turbine drivetrain system



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Fig. 14 Fault tree diagram of a wind turbine generator system

Table 7 Failure rates of the basic events for a wind turbine	Basic / intermediate ev	vent		Failure rate (h ⁻¹)
generator system	Mechanical failure	Bearing generator failure	Abnormal Vibration A	2.14×10^{-6}
			Asymmetry	5.85×10^{-6}
			Abnormal vibration B	2.14×10^{-6}
			Structural deficiency	1.17×10^{-6}
		Rotor and stator failure	Broken bars	2.10×10^{-7}
			Parameters deviation	1.63×10^{-5}
			Abnormal vibration C	2.14×10^{-6}
			Sensor failure	7.08×10^{-6}
			Temperature above limit	7.20×10^{-7}
	Electrical failure	Wire fault		1.00×10^{-7}
	~	Synchronization failure		3.61×10^{-6}

The gearbox is one of the most failure prone components 463 within the drivetrain system. Some of the major causes of 464 gearbox failure include: bearing and gear defects that result 465 from wear, excessive pressure, pitting, fatigue, gear tooth 466 deterioration, poor design of teeth, and poor material quality. 467 Another important factor which may significantly impact the 468 functioning of a gearbox is poor lubrication. Poor lubricant 469 quality, presence of dirt and debris, and problems in filter 470 can cause severe malfunction to rotating parts of the gearbox 471 system, and eventually lead to a sudden failure. The fault 472 tree diagram of the gearbox system is represented in Fig. 15. 473

The failure rates of the basic events for a wind turbine 474 gearbox system are given in Table 8. The potential dam-475 ages to the brake unit can cause the drivetrain system to 476 fail. Oil leakage, damage to brake disk, extreme loads that 477 can lead to overpressure, cracks on high-speed shaft and 478 brake overheating are considered to be the primary causes 479 for the brake unit failure. The fault tree diagram of a wind 480 turbine brake unit is shown in Fig. 16. The data for the 481 construction of this fault tree were collected from different 482 sources, e.g., [18, 37]. 483



Fig. 15 Fault tree diagram of a wind turbine gearbox system

 Table 8
 Failure rates of the basic events for a wind turbine gearbox system

Basic / intermediate eve	ent	Failure rate (h ⁻¹)
Bearings failure	Wear of bearings	1.00×10^{-5}
	Fatigue	3.00×10^{-7}
	Excessive pressure	1.00×10^{-6}
Insufficient lubrication	Abnormal filter	1.80×10^{-6}
	Debris (dirt)	2.14×10^{-6}
	Poor lubricant quality	1.80×10^{-6}
Gear failure	Abnormal vibration	2.14×10^{-6}
	Pitting/fatigue in gears	1.30×10^{-6}
	Gear tooth deterioration	3.00×10^{-7}
	Tooth surface defects	3.00×10^{-7}
	Poor design of teeth	1.00×10^{-6}
Overheating	Temperature above limit	7.08×10^{-6}
	Temperature sensor failure	7.20×10^{-7}

484 Pitch system

The pitch system controls the orientation of the turbine blades 485 486 in relation to the wind. The major contributors to pitch system failure include hydraulic system failure, wrong blade angle, 487 and drive alarm failure. Leakage in the hydraulic system, 488 489 overpressure and hydraulic motor failure are the major root causes for hydraulic system failure. The pitch system may fail 490 as a result of wrong blade angle, which in turn is caused by 491 meteorological unit failure. The meteorological unit provides 492 necessary wind data to the wind turbine control system. The 493 most common failures to the meteorological unit include dam-494 ages to the wind vane and anemometer. Figure 17 shows the 495 fault tree diagram of a wind turbine pitch system. The failure 496

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rates of the basic events for a wind turbine pitch system are 497 given in Table 9. 498

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FMEA of OC3 spar-type FOWT

An FMEA was performed on the OC3-Hywind spar-type 500 FOWT model to assess the criticality of different failure events 501 identified by the FTA method. In a similar fashion to FTA, 502 the FOWT components included in the FMEA study were 503 spar-buoy platform, mooring system, tower structure, blade 504 system, yaw system, drivetrain system (consisting of gearbox, 505 generator, and the brake unit), electronic components, pitch 506 system and hydraulic system. For each of these components, 507 failure modes were designated, which can occur through some 508 failure mechanisms, and the effects of these failures on the 509 system were evaluated. The 3 factors of O, S and D for each 510 failure mode were determined by interviewing experts (includ-511 ing designers, wind turbine operators, inspectors, maintenance 512 technicians, etc.) using FMEA questionnaire. The fault diagno-513 sis and prognosis techniques include visual inspection, vibra-514 tion analysis, non-destructive testing (NDT), SCADA based 515 condition monitoring, structural health monitoring as well as 516 remote inspections using remotely operated vehicles, aerial 517 drones and underwater sonar technology. The results of the 518 FMEA study for the OC3-Hywind spar-type FOWT model 519 are presented in a worksheet format in Table 10. 520

Discussion

FTA

After analyzing the fault tree diagrams in Figs. 4and17, the 523 failure rates of different subsystems of the FOWT model 524

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Fig. 17 Fault tree diagram of a wind turbine pitch system

were obtained. The results of the analysis are reported in Table 11.

It can be seen from Table 11 that tower structure and mooring system with mean failure rates of respectively 1.35×10^{-4} and 1.25×10^{-4} (per h) are the most prone subsystems to failure. These components are followed by electronic components and pitch system with failure rates 531 of 1.15×10^{-4} and 1.10×10^{-4} per h. Since these subassemblies/components are connected together in series, 533 the total failure rate of the FOWT system is calculated by 534 summing up the failure rates of all the individual subassemblies. Therefore, 536

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Basic / intermediate	e event	Failure rate (h ⁻¹)
Hydraulic fault	Abnormal vibration A	2.14×10^{-6}
	Hydraulic motor failure	1.00×10^{-5}
	Leakage in hydraulic system	4.80×10^{-5}
	Overpressure in hydraulic system	3.00×10^{-5}
Wrong blade angle	Abnormal vibration B	2.14×10^{-6}
	Wind vane damage	7.00×10^{-6}
	Anemometer damage	1.80×10^{-5}
Drive alarm fault	Lighting protection fault	1.00×10^{-5}
	Limit switch fault	1.00×10^{-5}

$$^{537} \quad \lambda = \sum_{i} \lambda_{i}$$

539

540

538 where λ_i represents the failure rate of the subassembly i (= 1, 2, ...) and λ is the failure rate of the FOWT system.

The failure rate of the FOWT system was estimated to be 541 approximately 7.01×10^{-4} per h, indicating that the mean 542 time between system failures (MTBSF) is about 1426.7 h. 543 The MTBSF estimated in this study is approximately 20% 544 larger than the value reported in [18]. This difference 545 between the results can be explained as follows: 546

- In this study, some further failure modes with more 547 548 detailed basic causes were considered.
- This paper focused on spar-type floating platforms, . 549 whereas [18] studied the failure scenarios for a semi-550 551 submersible platform.
- The mooring system in this study was considered as an 552 individual component of the FOWT model as opposed to 553 554 [18] in which mooring system failure was incorporated into the FOWT platform system. 555

After identifying the most critical components that can 556 cause the FOWT system to fail, minimal cut sets were com-557 puted to determine the most critical failure events. Cut sets 558 are unique combinations of component failures that can 559 cause system failure. A cut set is said to be a minimal cut if, 560 when any basic event is removed from the set, the remaining 561 events collectively are no longer a cut set. The results for the 562 probability of failure of tower structure as well as mooring 563 system due to different basic events are given in Tables 12 564 565 and 13, respectively.

As can be seen, the damages from external environmen-566 tal conditions like heavy storms, strong wind or wave, and 567 fatigue are the most dominant causes contributing to the 568 tower failure. On the other hand, abnormal stress, anchor 569 failure and fairlead fatigue are the main three causes of 570 571 mooring system failure.

FMEA

(4)

The risk priority number value for each component was deter-573 mined by summing up the RPNs associated with its failure 574 modes. Table 14 presents the RPN values for different FOWT 575 subassemblies/components. 576

As can be seen, the mooring system has the highest RPN 577 value, indicating that the mooring lines can be critical for the 578 safety of FOWT systems. This is followed by rotor blades, 579 gearbox, and tower structure. Among the three failure modes 580 contributing to mooring system failure, the mooring line 581 breakage with a RPN value of 364 was the most dominant fail-582 ure mode. Among the failure events causing the rotor blades 583 system to fail, the blades' structural damage was identified as 584 the most critical failure mode. 585

The results obtained from the FMEA study for the OC3-586 Hywind spar-type FOWT model were compared with those 587 reported for bottom-fixed offshore wind turbines. The com-588 parisons were made based on RPN rankings obtained for all 589 components that both FOWT and bottom-fixed wind turbines 590 have in common. As an example, the results of a comparison 591 between this study and our earlier study [26] are presented in 592 Table 15. As Table 15 shows, both studies ranked the blade 593 system as well as generator in the same order. However, the 594 studies presented minor differences in some other components 595 such as gearbox and pitch system. The results obtained by both 596 FTA and FMEA techniques were also compared with each 597 other. The failure criticality rankings obtained by both tech-598 niques are presented in Table 16. 599

As can be seen from the results of the FTA and FMEA 600 techniques, it is clear that there are some agreements between 601 the results. However, some differences were also observed that 602 might be attributed to the difference between FTA and FMEA 603 methodologies. The FTA is known to incorporate the causes of 604 various failure modes, whereas the FMEA is mainly used for 605 a single failure analysis. In terms of robustness, the decision 606 as to which method to choose for performing failure analysis 607 depends greatly on the input information which is available. If 608 failure data such as probability of failure on demand (PFD) or 609 rate of occurrence of failures (ROCOF) are available, the FTA 610 technique would be a more robust approach for failure analysis 611 than the FMEA technique. However, in the absence of quan-612 titative failure data or when the quality of data is insufficient, 613 the FMEA technique would be a more helpful method to use 614 as it can incorporate qualitative information through avenues 615 like expert elicitation. 616

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lable 10 The FME	A study results for th	le OC3-Hywind spar-t	ype FOWT model									
Item (ID)	Function	Potential Failure	Severity of effect	-	Probability of occurre	suce	Ability to detect					
		Mode	Potential Effects	SEV	Potential Causes of 6 Failure	occ	Current Design Controls	DET	RPN (cause)	RPN (Failure mode)	RPN (compo- nent)	RANK
Mooring system	To keep the posi- tion of the float-	Mooring lines breakage	FOWT shutdown	4	Fatigue	Ś	Visual inspection; NDT	7	140	364	664	1
	ing platform	Fairlead failure			Wear	с.	Sonar	7	84			
		Anchor failure			Abnormal stress	Ś	Visual inspection	7	140			
					Fatigue	Ś	Visual inspection; NDT	7	140	220		
					Corrosion	Ś	Visual inspection	4	80			
			Ŝ		Joint failure	, A	Visual inspection; ROV	4	32	80		
					Scour		ROV/ Sonar	4	48			
Tower structure	To integrate the	Tower structural	Structural failure	4	Lightning strike	, N	Visual inspection	4	32	184	184	4
	nacelle to sub-	damage		5	Strong wind/waves	ŝ	Visual inspection	4	48			
	structure part				Resonance		Structural health monitoring	4	48			
					Welding defects	2	NDT test	7	56			
Floating platform	To support the FOWT	Structural damage	FOWT shut down	4	External objects	6	Visual inspection; ROV	4	32	32	64	6
		Loss of stability			Mooring failure	, N	Visual inspection; ROV	4	32	32		
Blade system	To capture wind	Rotor system failure	Reduction in or loss of power	ŝ	Damage to bear-	, m	Vibration measure- ments	4	36	123	273	2
			production		Crack in rotor hub	, m	Visual inspection	٢	63			
					Imbalance of	2	Vibration measure-	4	24			
					blades		ments					
		Blade structural			Lighting	2	Visual inspection	7	42	150		
		damage			Erosion	2	Visual inspection	2	42			
					Cracks	2	NDT test	5	42			
					Delamination	, N	Visual inspection	4	24			
Yaw system	To align WT with wind direction	Yaw motor failure	Reduction in power 5	2	Abnormal vibra- tion	ŝ	Vibration measure- ments	-	42	42	106	8
		Drive alarm failure			Switch failure	6	Visual inspection	4	16	32		
					Lightning unit failure	6	Warning system	4	16			
		Meteorological			wind vane damage	2	Visual inspection	4	16	32		
		unit failure			Anemometer dam-	6	Visual inspection	4	16			
					age							

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Table 10 (continue	(þ											
Item (ID)	Function	Potential Failure	Severity of effect		Probability of occurren	nce /	Ability to detect					
		Mode	Potential Effects	SEV	Potential Causes of C Failure		Current Design Controls	DET	RPN (cause)	RPN (Failure mode)	RPN (compo- nent)	RANK
Gearbox	To increase the	Gears failure	Shutdown of the	ю	Tooth wear 3		Visual inspection	4	36	123	195	3
	low-speed rota- tional speed		WT and loss of power		Erosion 2		Visual inspection	4	24			
	4				Abnormal vibra- 3 tion	J	Condition monitor- ing	٢	63			
		Bearing failure			Wear 3		Visual inspection	4	36	72		
		y			Fatigue 3	-	VDT test	4	36			
Generator	To convert mechanical	Mechanical failure	Shutdown of the WT and loss of	б	Abnormal vibra- 3 tion	-	Vibration measure- ments	٢	63	123	177	5
	energy to electri- cal energy		power		Bearing damage 3		Vibration measure- ments	4	36			
				5	Rotor/stator failure 2		Visual inspection	4	24			
		Electrical failure			Fail to synchronize 1	-	Varning system	4	12	54		
					Wire failure 2		Visual inspection	7	42			
Brake system	To decelerate or	Hydraulic system	Shutdown of the	4	Oil leakage 3		Visual inspection	1	12	20	52	10
	decrease the	failure	ΜT		Motor brake failure 2		Visual inspection	1	8			
	speed	Overheating			Temp. sensor fault 2	-	Varning system	4	32	32		
Electronic compo-	To integrate the	Mechanical fault	Shutdown of the	ю	Corrosion 3		Visual inspection	4	36	72	114	7
nents	WT into power		WT		Dirt 3		Visual inspection	4	36			
	nug	Electrical fault			Short/open circuit 2		Alarm system	7	42	42		
Pitch system	To pitch the rotor	Meteorological	Shutdown of the	ю	Wind vane damage 2		Visual inspection	4	24	48	135	6
failure	blade	unit failure	TW		Anemometer dam- 2 age		Visual inspection	4	24			
		Hydraulic system			Abnormal vibra- 3		Vibration measure-	4	36	87		
		failure			tion		ments					
					Fluid Leakage 5		Visual inspection	<u></u>	15			
					Hydraulic motor 3	7	Alarm system	4	36			
					fault							

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617 **Conclusions and future work**

In this study, a failure analysis was performed for a float-618 ing offshore wind turbine (FOWT) concept based on an 619 OC3-Hywind spar type of platform moored to the seabed 620 with three anchor piles. The floating platform supports the 621 NREL 5 MW reference wind turbine with a rotor diameter 622 of 126 m and a tubular tower. All major mechanical, elec-623 trical and structural subassemblies of the system, includ-624 ing spar-buoy platform, mooring lines, tower, blade sys-625 tem, yaw system, gearbox, generator, brake unit, electronic 626 components, pitch and hydraulic system were included in 627 the analysis. The failure analysis approach relied on two 628 well-established reliability engineering methodologies, 629 namely, fault tree analysis and failure mode and effects 630 631 analysis.

The most critical subassemblies of the FOWT system 632 were identified by constructing fault tree diagrams and 633 estimating the rate of occurrence of failures. Since the 634 failure data for the FOWT subassemblies were scarce, the 635 information was collected from the reports published by 636 industries as well as expert opinions. Based on the results, 637 the tower structure and mooring system were determined 638 as the most failure-prone components in the FOWT sys-639 tem. These components experienced failure rates of 640 1.35×10^{-4} /h and 1.25×10^{-4} /, which correspond to mean 641 time between failures of, respectively, 309 and 334 days. 642 Also, in order to identify the most critical failure modes 643 and causes of FOWT components, the minimal cut sets 644 were computed. The overall failure rate of the FOWT sys-645 tem was estimated to be approximately 7.01×10^{-4} per 646 hour, indicating that the system would fail about six times 647 per year. 648

In addition to the FTA analysis, an FMEA study was also 649 performed to assess the 'criticality' of different failure mech-650 anisms in the FOWT subsystems. The failure criticality was 651 evaluated based on an index called the risk priority number, 652 which is the product of severity (S), occurrence (O), and 653 undetectability (D) ratings. These 3 ratings were determined 654 based on four-point scales being adopted and widely used 655 for bottom-fixed wind turbines in the wind energy sector. 656 The results showed that the mooring system and rotor blades 657 cause the highest risk to the FOWT system, followed by the 658 gearbox and tower structure. Among different failure modes 659 contributing to mooring system failure, the mooring line 660 breakage was found to be the most dominant failure mode. 661 Similarly, among different failure events causing the rotor 662 blades system to fail, the blades structural damage was rated 663 as the riskiest failure mode. The results obtained from the 664 FMEA analysis for the FOWT system were compared with 665 those reported for bottom-fixed offshore wind turbines. The 666

Table 11 Failure rates of different FOWT subsystems

No	Subsystem	Failure rate (hour ⁻¹)
1	Spar-buoy platform	1.06×10^{-4}
2	Mooring system	1.25×10^{-4}
3	Tower structure	1.35×10^{-4}
4	Electronic components	1.15×10^{-4}
5	Rotor blades	4.52×10^{-5}
6	Yaw system	2.17×10^{-5}
7	Gearbox	2.21×10^{-5}
8	Generator	1.47×10^{-5}
9	Brake unit	0.62×10^{-5}
10	Pitch system	1.10×10^{-4}

Table 12 Probability of tower failure due to different basic events

Basic event	Probability of failure
Storm	1.65×10^{-7}
Strong waves/winds	1.50×10^{-7}
Fatigue	3.30×10^{-8}
Lighting strike	2.10×10^{-8}
Welding defects	2.10×10^{-8}
Resonance	1.50×10^{-8}

 Table 13
 Probability of mooring system failure due to different basic events

Basic event	Probability of failure
Abnormal stress	1.23×10^{-6}
Anchor failure	5.40×10^{-7}
Fairlead fatigue	5.10×10^{-7}
Mooring line fatigue	5.10×10^{-7}
Corrosion	3.00×10^{-7}
Transitional wear	3.00×10^{-7}
Friction chain wear	2.10×10^{-7}
Chain corrosion	1.50×10^{-7}
Extreme sea conditions	7.02×10^{-14}

RPN rankings obtained in our work were in good agreement 667 with the previous studies in the literature. 668

Comparing the results obtained from the FMEA study 669 with those obtained from the FTA, a good agreement was 670 observed for failure criticality rankings. However, some dif-671 ferences were also found between the results which mainly 672 are attributed to the difference between FTA and FMEA 673 methodologies. The FTA methodology has the capability of 674 incorporating the basic causes of various failure scenarios, 675 whereas the FMEA methodology is often used for a sin-676 gle random failure analysis. In addition, the FTA is suitable 677 in situations where some historical data such as probability 678

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Table 14 RPN values for FOWT subsystems

No	Subsystem	RPN	Rank
1	Spar-buoy platform	64	9
2	Mooring system	664	1
3	Tower structure	184	4
4	Electronic components	114	7
5	Rotor blades	273	2
6	Yaw system	106	8
7	Gearbox	195	3
8	Generator	177	5
9	Brake unit	52	10
10	Pitch system	135	6

Table 15 RPNs and ranks for each FOWT subsystem

Components	RPN	Our rank	[26]
Spar-buoy platform	64	9	_
Mooring system	664	1	-
Tower structure	184	4	1
Electronic components	114	7	_
Rotor blades	273	2	2
Yaw system	106	8	13
Gearbox	195	3	2
Generator	177	5	5
Brake unit	52	10	11
Pitch system	135	6	7

Table 16 Ranking comparisons between FTA and FMEA

A	FMEA
wer structure	Mooring system
ooring system	Rotor blades
ectronic components	Gearbox
tch system	Tower structure
ar-buoy platform	Generator
otor blades	Pitch system
earbox	Electronic components
w system	Yaw system
enerator	Spar-buoy platform
ake unit	Brake unit
	wer structure boring system ectronic components ch system ar-buoy platform tor blades earbox w system enerator ake unit

of failure on demand (PFD) or rate of occurrence of failures 679 are available. However, the FMEA is a helpful method to use 680 during the preliminary design stages of floating wind tech-681 nologies, i.e., when there is lack of quantitative failure data. 682 The work performed in this study can be extended to 683 other FOWT concepts developed by Hexicon (https://www. 684 hexicon.eu/) or Principle Power (https://www.principlep 685 686 owerinc.com/). In addition, upscaling the results to a wind

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far nar of t	m level can greatly increase the effectiveness of mainte- nce activities which are proposed and scheduled as a result the failure analysis performed.	68 68 68
Fun L01	ding Engineering and Physical Sciences Research Council, EP/ 4106/1, Mahmood Shafiee	69 69
Dec	laration	69
Con on t	flict of interest Authors declare that there is no conflict of interest his study.	69: 69:
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